

K. D.K. College of Engineering, Nagpur

Department of Mechanical Engineering

Subject: - Heat Transfer

Fifth Semester

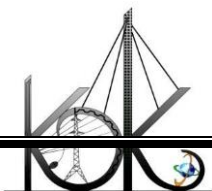
Assignment No. 1

- 1) Explain modes of heat transfer and Law of heat conduction.
- 2) Explain concept of Contact Resistant.
- 3) Critical thickness of insulation.
- 4) Derive three dimensional steady state heat conduction equation in Cartesian coordinate
- 5) Thermal contact Resistance.
- 6) Explain the different mode of heat transfer with their governing laws.
- 7) Derive the general heat conduction equation in cylindrical coordinates
- 8) Draw a neat sketch showing thermal resistances series and parallel. Write expression for heat transfer for both
- 9) Derive the general heat conduction equation with internal heat generation
- 10) Explain in brief with example the three modes of Heat Transfer
- 11) A certain building wall consists of 0.15 m of concrete [$K=0.15\text{W/m}\cdot\text{C}$], 0.15 m of fiber glass insulation and 10 mm of gypsum board [$K=0.15\text{ W/m}$]. The inside and outside convection coefficients are 10 and $40\text{W/m}^2\cdot\text{C}$, respectively. The outside air temperature is -6C and the inside temperature is 22C . Calculate the overall heat transfer coefficient for the wall the R value, and the heat loss per area.
- 12) A plane wall 80 mm thick ($K=0.15\text{ W/m}\cdot\text{C}$) is insulated on one side while the other is exposed to environment at 90C . The rate of heat generation within the wall is $12\times 10^4\text{ W/m}^3$. If the convective heat transfer coefficient between the wall and the environment is $560\text{ W/m}^2\cdot\text{C}$, determine the maximum temperature to which the wall will be subjected.
- 13) A Furnace wall consists of an inside layer of silica brick 15 cm thick. ($K=1.744\text{ W/mK}$) followed by a 30 cm thick later of magnesite brick. ($K=5.8\text{ W/mK}$) on the outside. The inside surface of silica wall is maintained at 780C while the outside surface magnesite is at 60C . The contact thermal resistance between tow walls at the interface is $2.57\times 10^{-3}\text{C/W}$ per unit wall area. What is the rate of heat loss through the composite structure per unit area of the wall? Also find the temperature drop at the interface. Draw a thermal circuit and find total resistance to heat flow.
- 14) An exterior wall of a house may be approximated by a 0.1 m layer of common brick ($k=0.7\text{ W/m}\cdot\text{C}$) followed by a 0.04 m layer of gypsum plaster ($k=0.40\text{ W/m}\cdot\text{C}$). What thickness of loosely packed rock wool insulation ($k=0.065\text{ W/m}\cdot\text{C}$) should be added to reduce the heat loss through the wall by 80%?
- 15) A wall is constructed of several layers. The first consists of brick ($K=0.66\text{ W/m}\cdot\text{K}$) 25 mm thick, the second layer consists of 2.5 cm thick mortar ($K=0.7\text{ W/m}\cdot\text{K}$), the third later of 10 cm thick Lime Stone ($K=0.66\text{W/m}\cdot\text{K}$) and the outer layer consists of 1.25 cm thick plaster ($K=0.7\text{W/m}\cdot\text{K}$). The heat transfer coefficient on the interior and exterior of the wall fluid layer are $5.8\text{W/m}^2\cdot\text{K}$ and $11.6\text{W/m}^2\cdot\text{K}$ respectively.

Find:-

- (a) Overall heat transfer coefficient from the air on the interior to the air at the exterior on the wall.
- (b) Overall thermal resistance per m^2
- (c) The rate of heat transfer per m^2 if the interior of the room is $26^\circ C$ while the outside air is at a temperature of $-7^\circ C$.
- (d) Temperature at the junction between the motor and lime stone.

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Assignment No. 2

- 1) A 2 kW resistance heat wire whose thermal conductivity ($K = 15 \text{ W/m}^\circ\text{C}$) has a diameter ($d = 4 \text{ mm}$) and length of ($L = 0.5 \text{ m}$) and is used to boil water. If the outer surface temperature of the resistance heat wire is 105°C , determine centre temperature of wire
- 2) A carbon steel ($K = 54 \text{ W/mK}$) rod with a cross section of an equilateral triangle (each side of 5 mm) is 1 cm long. It is attached to plane wall which is maintained at a temperature of 400°C . The surrounding environment is at 50°C and convective heat transfer coefficient is $90 \text{ W/m}^2\text{K}$. Calculate the heat dissipated by the rod
- 3) A wire of 6.5 mm diameter at a temperature of 60°C is to be insulated by a material having $k = 0.174 \text{ W/m}^\circ\text{C}$. Convection heat transfer coefficient (h_o) = $8.722 \text{ W/m}^2\text{C}$. The ambient temperature is 20°C . For maximum heat loss, what is the minimum thickness of insulation and heat loss per meter length? Also find percentage increase in heat dissipation
- 4) Determine the rate of heat flow through a spherical boiler wall which is 2 m in diameter and 2 m thick steel ($k = 58 \text{ W/m K}$). The outside surface of boiler wall is covered with asbestos ($k = 0.116 \text{ W/m K}$) 5 mm thick. The temperature of outer surface and that of fluid inside are 50°C and 300°C respectively. Take inner film resistance as 0.0023 K/W
- 5) Calculate the temperature distribution at the middle and rate of heat flow at the root of a turbine blade 80 mm long, 600 mm^2 in C.S. and 159 mm in perimeter. The blade is made of stainless steel ($k = 23.3 \text{ W/m k}$) and is exposed to steam at 1000°C . While its root is maintained at 600°C . The heat transfer coefficient between the blade surface steam is $5000 \text{ W/m}^2\text{K}$.
- 6) A hot cylinder ingot 50 mm diameter and 200 mm long is taken out from the furnace at 800°C and dipped in water till its temperature falls to 500°C . Then it is directly exposed to air till its temperature falls to 100°C . Find the total time required for the ingot to reach the temperature from 800 to 100°C . Take the following:
 $K = 60 \text{ W/m}^\circ\text{C}$, $C = 200 \text{ J/kg}^\circ\text{C}$, $\rho = 800 \text{ kg/m}^3$
 Heat transfer coefficient in water = $200 \text{ W/m}^2\text{C}$
 Heat transfer coefficient in air = $20 \text{ W/m}^2\text{C}$, Temperature of air = 30°C .
7. What is meant by a lumped capacity? What are the physical assumptions necessary for a lumped-capacity unsteady-state analysis to apply?
8. What is meant by a semi-infinite solid?
9. What initial conditions are imposed on the transient solutions presented in graphical form in this chapter?
10. What boundary conditions are applied to problems in this chapter?
11. Define the error function.
12. Define the Biot and Fourier numbers.



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Assignment No. 3

- 1) Water at the rate of 3 kg/s is heated from 5 to 15°C by passing it through a 50 mm ID copper tube. The tube wall temperature is maintained at 90°C . What is the length of the tube?

2) Air at 30°C is flowing over 2 cm long plate maintained at 70°C at m/s. Determine heat transfer from the plate.

3) A highly viscous fluid flows through a 5 cm I.D. pipe at rate 50 kg/hr. Fluid passes through 1 m long heated section where a constant flux of 1000 W/m² is supplied. Calculate the final temperature of liquid if initial temperature is 40°C. Obtain the maximum wall temperature. Assume properties of liquid as,

$$\rho = 1500 \text{ kg/m}^3,$$

$$C_p = 1.675 \text{ kJ/kg K},$$

$$k_s = 0.865 \text{ W/mK}.$$

4) Water at 20°C is to be heated by passing it through the tube. Surface of tube is maintained at 90°C. The diameter of tube is 4 cm while its length is 9 m. Find the mass flow rate so that exit temperature of water will be 60°C. The properties of water are

$$\rho = 995 \text{ kg/m}^3$$

$$C_p = 4.175 \text{ kJ/kg K},$$

$$k = 0.64 \text{ W/mK}, \quad \nu = 0.62 \times 10^{-6} \text{ m}^2/\text{s},$$

$$\beta = 4.25 \times 10^{-3} \text{ K}^{-1}$$

$$\text{Use the correlation } Nu = 0.023 \cdot (Re)^{0.8} \cdot (Pr)^{0.3}$$

5) In a certain process, castor oil at 30°C flows past a flat plate. The velocity of oil is 0.08 m/sec. The length of the plate is 5 m. The plate is heated uniformly and maintained at 90°C. Calculate the following:

(i) Hydrodynamic and thermal boundary layer thickness at the trailing edge of plate.

(ii) Total drag force per unit width on one side of the plate.

Use the following correlation:

$$Nu = 0.332 (Re)^{1/2} \cdot (Pr)^{1/3}$$

Take properties as,

$$\rho = 956.8 \text{ kg/m}^3, \quad k = 0.213 \text{ W/mK},$$

$$a = 7.2 \times 10^{-8} \text{ m}^2/\text{s}$$

$$\nu = 0.65 \times 10^{-4} \text{ m}^2/\text{s}$$

6) Explain boundary layer concept and define Hydrodynamic and thermal boundary layer with reference to flow over flow heated plate.

7) What do you mean by Lumped heat capacity analysis State the significance of Heisler chart.

8) In a certain process, castor oil at 30°C flows past a flat plate. The velocity of oil is 0.08 m/sec. The length of the plate is 5 m. The plate is heated uniformly and maintained at 90°C.

Calculate the following

(a) Hydrodynamic and Thermal Boundary layer thickness at the trailing edge of plate.

(b) Total drag force per unit width on one side of the plate Use following correlation:-

$$Nu = 0.332 (Re)^{1/2} (Pr)^{1/3}$$

Take properties as,

$$\rho = 956.8 \text{ kg/m}^3, \quad k = 0.2132 \text{ W/mK}, \quad a = 7.2 \times 10^{-8} \text{ m}^2/\text{s}, \quad \nu = 0.65 \times 10^{-4} \text{ m}^2/\text{s}$$

9) Air at 160°C pressure of 1 bar is flowing over a plate at a velocity of 3 m/s. If the plate is 30 cm wide and at 60°C, using exact method at $x=30$ cm at the distance corresponding to the transition point, Find the following:

(1) Boundary layer thickness

(2) Average friction coefficient

(3) Shearing stress due to friction

(4) Average convection heat transfer coefficient

(5) Total drag force on the plate

Take properties at Bulk mean temperature as,

$$\rho = 1.1374 \text{ kg/m}^3, \quad k = 2.732 \times 10^{-2} \text{ W/mK}$$

$$C_p = 1.005 \text{ kJ/kg K}, \quad \nu = 16 \times 10^{-6} \text{ m}^2/\text{sec}$$

Use correlation as, $pr = 0.7$

$$Nu = 0.332 (Re_x)^{1/2} (Pr)^{1/3}$$

10) Explain the following Dimensionless number and their physical significance:

- (i) Reynolds number
- (ii) Prandtl number
- (iii) Nusselt number

11) In a particular solar collector of cylindrical type, energy is collector by placing a tube at the focal line of parabolic collector and passing fluid through the tube. The arrangement resulting is a uniform heat flux of 2000 W/m^2 along the axis of the tube of diameter 60 mm. Determine:

- (i) Length of the tube required to the heat water from 20°C to 80°C which flows at the rate of 0.01 kg/s .
- (ii) Surface temperature at the outlet

Properties of water are:

$$u = 352 \times 10^{-6} \text{ Ns/m}^2, C_p = 4187 \text{ J/kg K}$$

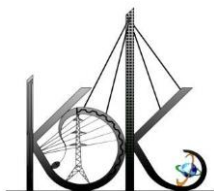
$$k = 0.67 \text{ W/m K.}$$

12) Air at a velocity of 3 m/s and at 29°C flows over a flat plate along its length. The length, width and thickness of the plate are 100 cm and 50 cm respectively. The top surface of the plate is maintained at 100°C . Calculate the heat loss by the plate and steady state conditions. The thermal conductivity of the plate may taken as 23 W/mK .

(b) Write short notes on (any two):

- (i) Biot number and Fourier and their significance.
- (ii) Hydrodynamic and Thermal boundary layer
- (iii) Lumped heat capacity and Heisler charts.

13) Air at a velocity of 3 m/sec and 20°C flows over a flat plate along its length. The length, width and thickness of the plate are 100 cm , 50 cm and 2 cm respectively. The top surface of the plate is maintained at 100°C . Calculate the heat loss by the plate and bottom temperature of the plate for the steady state condition. Conductivity of the plate may be taken at 23 W/mo K .



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Assignment No. 4

1) Sketch temperature and velocity profiles in free convection on a vertical wall:-

- i) When the plate is being cooled.
- ii) When the plate is heated

2) Calculate the heat transfer from a 60 W incandescent bulb at 115°C to ambient air 25°C. Assume the bulb as a sphere of 50 mm diameter. Also find the percentage of power lost by free convection.

3) Explain briefly the various regions in boiling heat transfer.

4) A vertical tube of 60 mm outside diameter and 2.2 m long is exposed to steam at atmospheric pressure. The outer surface of the tube is maintained at a temperature of 50°C by circulating cold water through the tube. Calculate-i) Rate of heat transfer to the coolant.ii) Rate of condensation of steam.

Take properties as:-

$$\rho = 975 \text{ kg/m}^3, \mu = 375 \times 10^{-6} \text{ N-s/m}^2$$

$$k = 0.67 \text{ W/m}^\circ\text{C}, \rho_v = 0.59 \text{ kg/m}^3.$$

$$h_{fg} = 2257 \text{ kJ/kg}.$$

5) A 2 – stroke cycle petrol engine cylinder consists of 16 fins. If the surface temp. is 475°C and atmospheric air temperature is 25°C, calculate the heat transfer rate from the fins for the following cases :

(a) When the motor cycle is stationary.

(b) When the motor cycle is running at a speed of 60 km/h.

The fin may be idealized as single horizontal flat plate of the same area.

Properties of air at 25°C

$$k = 4.266 \times 10^{-2} \text{ W/m}^\circ\text{C}, \nu = 40.61 \times 10^{-6} \text{ m}^2/\text{s}, Pr = 0.677$$

Use correlations :

$$Nu = 0.54 [Gr \cdot Pr]^{0.25} \text{ for laminar flow}$$

$$Nu = 0.036 [Re]^{0.8} [Pr]^{0.33} \text{ for turbulent flow.}$$

6) A steam condenser consisting of a square array of 625 horizontal tubes, each 6 mm in diameter, is installed at the exhaust of a steam turbine. The tubes are exposed to saturated steam at a pressure of 15 kPa. If the tube surface temperature is maintained at 25°C, calculate :

i) The heat transfer coefficient and

ii) the rate at which steam is condensed per unit length of the tubes.

Assume film condensation on the tubes and absence of non-condensable gases.

Properties of saturated water at film temperature are

$$\rho = 992 \text{ kg/m}^3, \mu = 663 \times 10^{-6} \text{ N-s/m}^2, k = 0.0631 \text{ W/m}^\circ\text{C}.$$

7) What is the significance of Grashof's Number and Rayleigh Number.

8) Explain with neat sketch the pool boiling curve and Regime of pool boiling.

9) A hot plate of 40 cm in height and 70 cm wide at 150°C is exposed to the air at 30°C. Calculate the following:-

- (i) Maximum velocity at 30 cm from the leading edge of the plate.
- (ii) Mean velocity at 30 cm from the leading edge.
- (iii) Boundary layer transfer thickness at 30 cm from the leading edge.
- (iv) Local heat transfer coefficient at 30 cm from the leading edge.
- (v) Average heat transfer coefficient.
- (vi) Total mass flow through the boundary.
- (vii) Total heat loss from the side of the plate.
- (viii) Rise in temperature of the air passing the boundary.
- (ix) Total drag force on the plate.

10) State the effect of non-condensable gases on condensation.

11) Explain with neat sketch development of velocity boundary layer on hot and cold vertical plate subjected to Natural Convection.

12) A horizontal fluorescent tube which is 3.8 cm in diameter and 120 cm long stands in still air at 1 atm. and 20°C. If the surface temperature of the tube is 40°C and radiation is neglected, what percentage of power is being dissipated by convection? Take properties of air as $\nu = 16.19 \times 10^{-6} \text{ m}^2/\text{sec}$, $K_{\text{air}} = 0.02652 \text{ W/mK}$, $Pr = 0.706$, $\rho = 1.02 \text{ kg/m}^3$, $C_p = 1.004 \text{ kJ/kg K}$

Use relation $Nu_D = 0.53 [Ra_D]^{1/4}$

13) Air flow through a long rectangular (30 cm height X 65 cm width) air conditioning the outer duct surface temperature at 15°C. If the duct is insulated and exposed to air at 25°C, calculate the heat gained by the duct per meter length assuming it to be horizontal. Use the following correlation:

Vertical surface = $Nu_L = 0.59 (Ra_L)^{1/4}$

Upper surface heated lower surface

Cooled – $Nu_L = 0.54 (Ra_L)^{1/4}$

Lower surface heated or upper surface

Cooled – $Nu_L = (Ra_L)^{1/4}$

Taking the following properties of air:

$\rho = 1.205 \text{ kg/m}^3$; $C_p = 1005 \text{ J/kg K}$

$\nu = 15 \times 10^{-6} \text{ m}^2/\text{sec}$; $K = 0.02593 \text{ W/mK}$

14) Saturated F-12 vapour at 12.3/bar condenses on the outside of a bank of horizontal tubes 1 cm OD arranged in 5X5 square array. Calculate the rate of condensation per meter length of the array if the tube surface is maintained at 40°C.

F-12 has following properties:-

Liquid density = 1218 kg/m³, Latent heat = 128.12 kJ/kg, Liquid K = 0.0686 W/mo K, Liquid μ = 2.84 X 10⁻⁶ kg/m-s.

Saturation temperature at 12.3/bar = 48.9°C

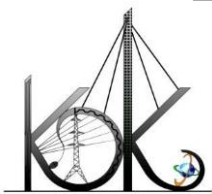
15) A vertical plate is heated from one side and is maintained at 96°C. On the other side is air at 30°C. Calculate:-

(i) Local value of convective Heat Transfer coefficients at distance of 20 cm from the lower edge.

(ii) Average value of convective Heat transfer coefficient over the whole 20 cm length of plate.

16) A steel plate 20 cm² and 0.5 cm thick heated uniformly to 430°C and is kept vertically in still air at a temperature of 20°C. Neglecting radiation or making suitable assumption. Calculate time required for the plate to cool at 330°C.

17) Explain in brief Heat Pipe.



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Assignment No. 5

1) A double walled flask may be idealized to be equivalent to two infinite parallel plates. The emissivities of wall are 0.3 and 0.7 respectively. The space between them is evacuated. A shield of polished Aluminium of $\epsilon = 0.05$ is inserted between them. Find the reduction in heat transfer rate due to insertion of radiation shield.

2) Explain the following:

(i) Kirchhoff's law of radiation,

(ii) Wien's displacement law,

iii) Shape factor

3) Explain shape factor and calculate the shape factor for a cylindrical cavity of diameter D and height L with respect to itself.

4) An industrial furnace in the form of a black body emits radiation at 3000K calculate:-

i) Monochromatic emissive power at $1\ \mu\text{m}$ wavelength.

ii) Wavelength at which the emission is maximum.

iii) Maximum emissive power

5) The net radiation from the surface of two parallel plates maintained at T_1 and T_2 is to be reduced by 99%. Calculate the number of screens to be placed between the two surfaces to achieve this reduction in heat exchanger assuming the emissivities of the screens as 0.05 and that of surface as 0.8

6) Two parallel square plates, each 4m^2 area are large, compared to a gap of 5mm separating them. One plate has a temperature of 800K and surface emissivities of 0.6, while the other has a temperature of 300K and surface emissivities of 0.9 find the net energy exchange by radiation between the plates. If a thin polished metal sheet of surface emissivities 0.1 on both sides is now located centrally between the two plates, what will be its steady state temperature? How the heat transfer would be altered? Neglect the convection and edge effects, if any. Comment upon the significance of this exercise

7) An enclosure measures $1.5 \times 1.7\text{m}$ with a height of 2m . The walls and ceiling are maintained at 250°C and the floor at 130°C . The walls and ceiling have an emissivity of 0.82 and the floor 0.7. Determine the net radiation to the floor.

8) Two very large parallel planes with emissivities 0.3 and 0.8 exchange radiative energy. Determine the percentage reduction in radiative energy transfer when a polished aluminum radiation shield ($\epsilon = 0.04$) is placed between them.

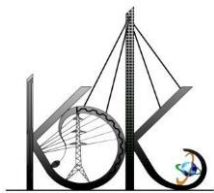
9) Determine the net radiation heat exchanger between two surfaces:-

$$A_1 = (\epsilon_1 = 0.8, T_1 = 1000\text{K})$$

$$A_2 = (\epsilon_2 = 1.0, T_2 = 500\text{K})$$

Neglect radiation from the surrounding. Take Shape factor $F_{12} = 0.22$

10) Liquid oxygen at atmospheric pressure (normal boiling point -180°C) is to be stored in a cylinder container which can be considered as a sphere of 0.3m outer diameter. The system is insulated by enclosing the container inside another concentric sphere of 0.45m diameter with intermediate space evacuated. Both the sphere surfaces are made of Aluminium ($\epsilon = 0.3$). The outer sphere has a temperature of 40°C . Estimate the rate of heat transfer flow by radiation to oxygen. What will be the reduction in heat gained if polished Aluminium with emissivity of 0.03 is used?



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Assignment No. 6

- 1) A parallel flow of heat transfer uses 1500 kg/hr of cold water entering at 25°C to cool 600 kg/hr of Hot water at 70°C. The exit temperature on hot side is required to be 59°C. Neglecting the effects of fouling factor, calculate the area of Heat exchanger. It may be assumed that the individual Heat Transfer coefficient on both sides are 1600 W/m²K. Use mean temperature difference and NTU approach separately. Also calculate the exit temperature of hot and cold steam if the flow of hot water is doubled, i.e. 1200 kg/hr. It has been stated that individual Heat Transfer coefficient are proportional to 0.8th power of the flow rate. Take $C_p = 4180 \text{ J/kgK}$.
- 2) In an oil cooler, oil enters at 160°C. If water entering at 35°C flows parallel to oil, the exit temperature of oil and water are 90°C and 70°C respectively. Determine the exit temperature of oil and water if the two fluids flow in opposite directions. Assume that the flow rates of the two fluids and U_o remain unaltered. What would be the minimum temperature to which oil could be cooled in parallel flow and counter-flow operations?

3) A tubular heat exchanger consists of 1200 tube each 20 mm OD and 5m length. Hot fluid flows inside the tube and cold over it but in opposite direction to hot fluid. The overall heat transfer coefficient based on OD is $320 \text{ W/m}^2 \text{ K}$. Determine the outlet temperatures of both fluids and total heat transfer. Given, $T_{hi} = 120^\circ\text{C}$, $T_{ci} = 20^\circ\text{C}$, $m_h = 20 \text{ kg/s}$, $m_c = 5 \text{ kg/s}$, $C_{pc} = 4000 \text{ J/kg K}$, $C_{ph} = 2000 \text{ J/kg K}$.

4) A shell and tube type steam condenser, employed in a large steam power plant, effects a heat exchange rate of 2200 MW. The condenser of a single pass shell and 32000 tubes, each exceeding two passes. The water at the rate of $3.2 \times 10^4 \text{ kg/sec}$ passes through the tubes which are of thin wall and of diameter 30 m. the steam condenser on the outer surface on the tubes. The heat transfer coefficient on the steam side may be taken as $11500 \text{ W/m}^2\text{K}$. Steam condenses at 50°C while water enters the condenser at 20°C . Using LMTD correction factor method and NTU method, calculate the following:-(i) Outlet temperature of cooling water.(ii) Length of tube per pass, Take following properties of water: $C_p = 4.18 \text{ kJ/ kgo K}$, $\mu = 855 \times 10^{-6} \text{ N-sec/m}^2$, $K = 0.613 \text{ W/mo K}$, $Pr = 5.83$.

5) A counter flow heat exchanger is employed to cool 0.55 kg/s ($C_p = 2.45 \text{ kJ/kgo C}$) of oil from 155°C to 40°C by the use of water. The inlet and outlet temperature of cooling water are 15°C and 75°C respectively. The overall heat transfer coefficient is expected to be $1450 \text{ W/m}^2\text{C}$. Using NTU method calculate the following :i) The mass flow rate of water

ii) The effectiveness of heat exchanger, iii) The surface area required.

6) An oil cooler of the from tubular heat exchanger cools oil from a temperature of 85°C to 35°C by a large pool of stagnant water assumed at constant temperature of 25°C . The tube carrying oil is 35 m long and 25 mm inside diameter. The specific heat and specific gravity of oil are 2.51 kJ/kg oK and 0.8 respectively. The average velocity of the oil is 60 cm/sec . Estimate the overall heat transfer coefficient obtained from the system

7) Cold water at 1495 kg/h enters at 25°C through a parallel flow heat exchanger to cool 605 kg/h of hot water entering at 70°C and leaving at 50°C . Find the area of heat exchanger. The individual heat transfer coefficients on both sides are $1590 \text{ W/m}^2\text{K}$. Use LMTD and NTU method. Find also the exit temperature of cold and hot fluid streams, if the flow of hot water is doubled. Assume that the individual heat transfer coefficients are proportional to 0.8th power of the flow rate.

For water $C_p = 4180 \text{ J/kg K}$.

8) In an industry, 0.6 hg/sec of oil, ($C_p = 2.5\text{kJ/kgok}$) is to be cooled in a counter flow heat exchanger from 110°C to 35°C by the use of water entering at 20°C . The overall heat transfer coefficient is expected to be $1500 \text{ W/m}^2\text{oK}$. Presume that the exit temperature of water is not to exceed 80°C .Using NTU Method, calculate,

(1) Water flow rate, (2) Surface area required,

(3) Effectiveness of exchanger

9) The flow rates of hot and cold water streams running through a parallel flow heat exchanger are 0.2 kg/sec and 0.5 kg/sec respectively. The inlet temperatures on the hot and cold side are 75°C and 20°C respectively. The exit temperature of hot water is 45°C . If the individual heat transfer coefficients on both sides are $650 \text{ W/m}^2\text{oC}$, calculate area of heat exchanger.

10)) In a food processing plant a brine solution is heated from 6oC to 12oC in a double pipe that exchanger by water entering at 50oC and leaving at 40oC at the rate of 0.166 kg/s. if the overall hate transfer coefficient is 850 W/m², what heat exchanger area is required for

i) Parallel flow

ii) Counter flow.

11. Define the overall heat-transfer coefficient.

12. What is a fouling factor?

13. Why does a “mixed” or “unmixed” fluid arrangement influence heat-exchanger performance?

14. When is the LMTD method most applicable to heat-exchanger calculations?

15. Define effectiveness.

16. What advantage does the effectiveness-NTU method have over the LMTD method?

17. What is meant by the “minimum” fluid?

18. Why is a counterflow exchanger more effective than a parallel-flow exchanger?

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